## FLUID MECHANICS

## TESTS

Attention: there might be more correct answers to the questions.

## Chapter 10: Hydraulics

T.10.1.1 Real fluid flows in a pipeline that consist of straight and bent pipes, diffusers etc.
a, Pressure always decreases in the direction of the flow.
$b$, Pressure might increase in the direction of the flow.
c, the Bernoulli-sum always increases in the direction of the flow.
d, the Bernoulli-sum always decreases in the direction of the flow.
The answer is:
T.10.1.2 Using dimensional analysis
a, the number of dimensioned physical quantities ( $n$ ) influencing the problem can be decreased.
$b, n+r$ nondimensional groups can be created, where $r$ is the rank of the dimension matrix
$c, n-r$ nondimensional groups can be created
$d$, the type of dependence (power or trigonometric function etc.) between the nondimensional variables can be determined

The answer is:
T.10.2.1 Assuming a filled rectangular pipe cross-section of the size $a \times b$ the pipe friction loss can be determined using the equivalent diameter, $d_{e}$ calculated as:
$a, \frac{d_{e} \pi}{4}=a b$
$b, d_{e} \pi=2(a+b)$
c, $d_{e}^{2}=a^{2}+b^{2}$
$d, d_{e}=\frac{4 a b}{2(a+b)}$
$e$, None of the above are correct.
The answer is:
T.10.2.2 In a turbulent pipe flow the pipe friction coefficient $\lambda$ is the same for smooth and rough pipes if
a, the pipe friction coefficient does not depend on the Reynolds number
$b$, if the wall roughness size is smaller than the viscous sublayer thickness
c, if the wall roughness size is greater than the viscous sublayer thickness
$d$, if the Reynolds number is $\mathrm{Re}>10^{5}$
$e$, None of the above are correct.
The answer is:
T.10.3.1When the flow separates from the diffuser wall
a, losses are greater than in the non-separated case
$b$, losses are smaller than in the non-separated case
Separation can be avoided by
$c$, decreasing the diffuser angle
d, increasing the diffuser angle
$e$, Losses do not depend on the diffuser angle.
The answer is:
T.10.3.2 Choose the correct statements!

|  | pressure | mean velocity |
| :--- | :---: | :---: |
| Diffuser inlet cross section | $p_{\text {in }}$ | $v_{\text {in }}$ |
| Diffuser outlet cross section |  |  |
| - inviscid | $p_{\text {out }}$ | $v_{\text {out }}$ |
| - viscous | $p_{\text {outv }}$ | $v_{\text {outv }}$ |

a, $p_{\text {in }}<p_{\text {out }}$ and $p_{\text {outv }}<p_{\text {out }}$
$b, p_{\text {in }}<p_{\text {out }}$ and $p_{\text {outv }}>p_{\text {out }}$
c, $v_{\text {in }}<v_{\text {out }}$ and $v_{\text {outv }}<v_{\text {out }}$
$d, v_{\text {in }}>v_{\text {out }}$ and $v_{\text {outv }}>v_{\text {out }}$
$e$, None of the above are correct.
The answer is:
T.10.4.1 Which of the following formulae are correct in case of a hydraulically smooth pipe?
a, If $\mathrm{Re}=2.3 \cdot 10^{4}$ then $\lambda=\frac{64}{\mathrm{Re}}$.
b, If $\operatorname{Re}=2.3 \cdot 10^{4}$ then $\lambda=\frac{0.316}{\sqrt[4]{\operatorname{Re}}}$.
c, If $\operatorname{Re}>2.3 \cdot 10^{5}$ then $\lambda=\frac{0.316}{\sqrt[4]{\mathrm{Re}}}$.
d, If $\operatorname{Re}<2.3 \cdot 10^{3}$ then $\lambda=\frac{64}{\operatorname{Re}}$.
e, If $\mathrm{Re}=5 \cdot 10^{5}$ then $\lambda=\frac{0.316}{\sqrt[4]{\operatorname{Re}}}$.
The answer is:
TZ.10.1 Friction loss in a pipe in case of a turbulent flow
$a$, is roughly directly proportional to the mean velocity
$b$, is inversely proportional to the mean velocity squared
c, is roughly inversely proportional to the diameter squared
d, depends on the position of the pipe
$e$, is roughly proportional to the mean velocity squared
The answer is:

TZ.10.2Hydraulically equivalent diameter
$a$, is the ratio of the wetted perimeter over the cross-section
$b$, is the ratio of cross-section over the square root of wetted perimeter
c, is the ratio of two times the cross-section over the wetted perimeter
d, is the ratio of four times the cross-section over the wetted perimeter
$e$, None of the above are correct.
The answer is:
TZ.10.3 Friction loss in a rough pipe is $\Delta p_{r}^{\prime}$. In a smooth pipe of the same size, it is $\Delta p_{s}^{\prime}$. The flow rate $q_{v}$, pipe length and the fluid are the same.
a, $\Delta p_{r}^{\prime}>\Delta p_{s}^{\prime}$ for all $q_{v}$
$b, \Delta p_{r}^{\prime}$ can be smaller than $\Delta p_{s}^{\prime}$
c, $\Delta p_{r}^{\prime}$ can be equal to $\Delta p_{s}^{\prime}$
d, $\Delta p_{r}^{\prime}$ can be greater than $\Delta p_{s}^{\prime}$
$e, \frac{\Delta p_{r}^{\prime}}{\Delta p_{s}^{\prime}}$ is constant for all $q_{v}$
The answer is:
TZ.10.4 Choose the correct statements!

|  | pressure | mean velocity |
| :--- | :---: | :---: |
| Diffuser inlet cross section | $p_{\text {in }}$ | $v_{\text {in }}$ |
| Diffuser outlet cross section |  |  |
| - inviscid | $p_{\text {out }}$ | $v_{\text {out }}$ |
| - viscous | $p_{\text {outv }}$ | $v_{\text {outv }}$ |

Diffuser efficiency $\eta_{\text {diff }}$ is calculated as:
a, $\frac{p_{\text {outv }}-P_{\text {in }}}{p_{\text {out }}-p_{\text {in }}}$
b, $\frac{\frac{\rho}{2}\left(v_{\text {in }}^{2}-v_{\text {outv }}^{2}\right)}{\frac{\rho}{2}\left(v_{\text {in }}^{2}-v_{\text {out }}^{2}\right)}$
$c, p_{\text {out }}-p_{\text {outv }}=\left(1-\eta_{\text {diff }}\right)\left(p_{\text {outv }}-p_{\text {in }}\right)$
d, $\frac{p_{\text {outv }}+\frac{\rho}{2} v_{\text {outv }}^{2}}{p_{\text {in }}+\frac{\rho}{2} v_{\text {in }}^{2}}$
$e$, None of the above are correct.

