

SOCRATES Teaching Staff Mobility Program
2000 - 2001

Technical University of Budapest



1782

Department of Fluid Mechanics



DMA-URLS

Finite Element Method for Turbomachinery Flows

Test Cases

Alessandro Corsini

Dipartimento di Meccanica e Aeronautica, University of Rome "La Sapienza"

BUDAPEST University of Technology and Economics - 28 November 2000

PARALLEL RESULTS

□ EVALUATION OF PARALLEL SOLVER PERFORMANCES

$$\textit{Speed-up Factor} \quad S(pr) = T(pr_{ref}) / T(pr)$$

$$\textit{Efficiency} \quad E(pr) = S(pr) / (pr / pr_{ref})$$

pr = processors number

T = elapsed time

pr_{ref} = reference processors numbers

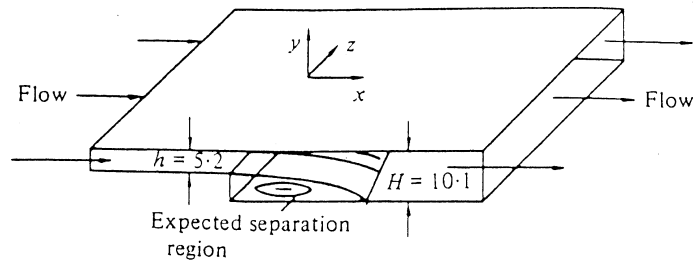
□ CODE SETTING PARAMETERS

- MIXED ORDER FEM INTERPOLATION FUNCTIONS (Q2/Q1)
- STANDARD K - \mathbf{e} MODEL
- GMRES(50), GAUSS-SIEDEL PRECONDITIONER
- *CONDENSED* CYCLES CONFIGURATION

□ CONVERGENCE CRITERIA

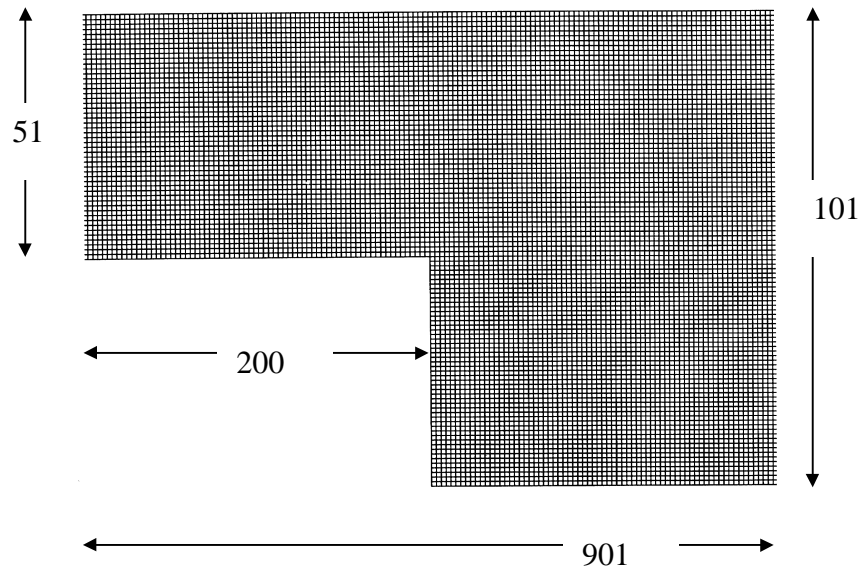
$$R_{res} = \|r_k\|_2 / \|r_0\|_2 = 10^{-4}, \quad R_{sol} = (\|U_k\|_2 - \|U_{k-1}\|_2) / \|U_k\|_2 = 10^{-4}$$

LAMINAR FLOW IN A BACKWARD FACING STEP



- EXPERIMENTS ARMALY ET AL. (1983)
- RE=300
- CRAY T3E

□ GRID DETAIL (81001 GRID POINTS)

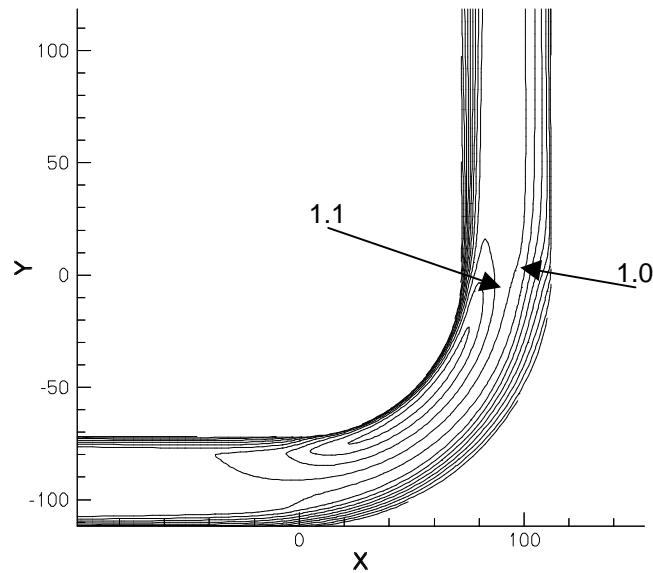


Domains	Reattachment point coordinate (x_s/h)
4	6.323
16	6.324
32	6.320
Experiments	6.326

Domains	$S(pr)^*4$	$E(pr)$
4	4.00	1.00
16	15.38	0.96
32	21.63	0.68

COMPARATIVE ANALYSIS OF PARALLEL SOLUTION TECHNIQUES (ii)

TURBULENT FLOW IN A TWO-DIMENSIONAL 90° BEND



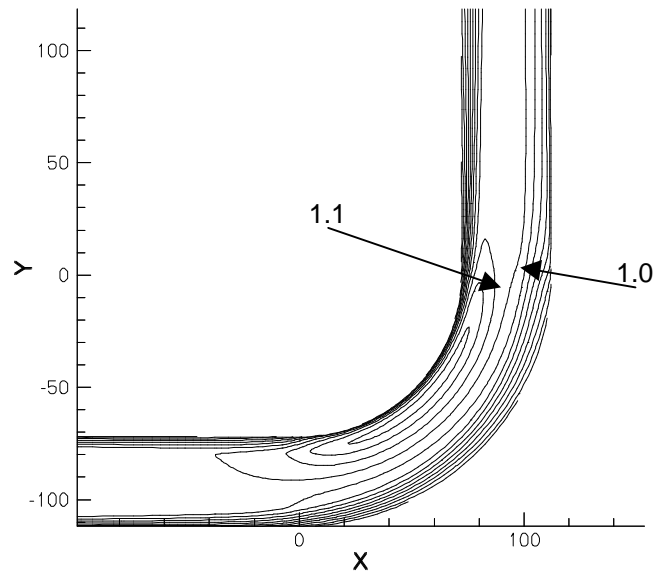
Computed velocity contour

- RE=40.000
- 19521 GRID POINTS (241×81)
- GMRES(50)
- CONVERGENCE THRESHOLD $R_{res}, R_{sol} = 10^{-4}$

Domains	$n_{it}=n_{tot}$ Condensed cycles	n_{it} Nested cycles	n_{tot} Nested cycles
2	250	201	804
3	234	151	604
6	228	138	552

TURBULENT FLOW IN A TWO-DIMENSIONAL 90° BEND

- RE=40.000
- 19521 GRID POINTS (241×81)
- IBM SP2

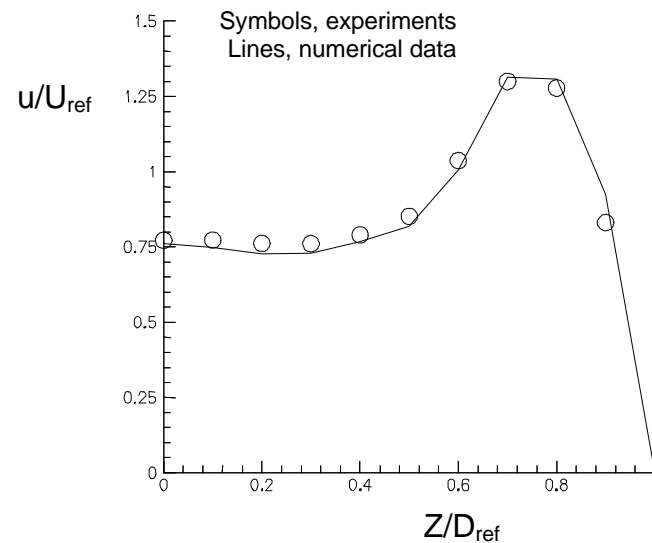


Computed velocity contour

<i>Domains</i>	<i>S(pr)</i>	<i>E(pr)</i>
2	2.00	1.00
3	3.16	1.05
6	6.28	1.05

LAMINAR FLOW IN A 90° THREE-DIMENSIONAL ELBOW

- EXPERIMENTS TAYLOR ET AL. (1981)
- $RE=790$
- GRID (74151 POINTS, $321 \times 21 \times 11$)
- CRAY T3E



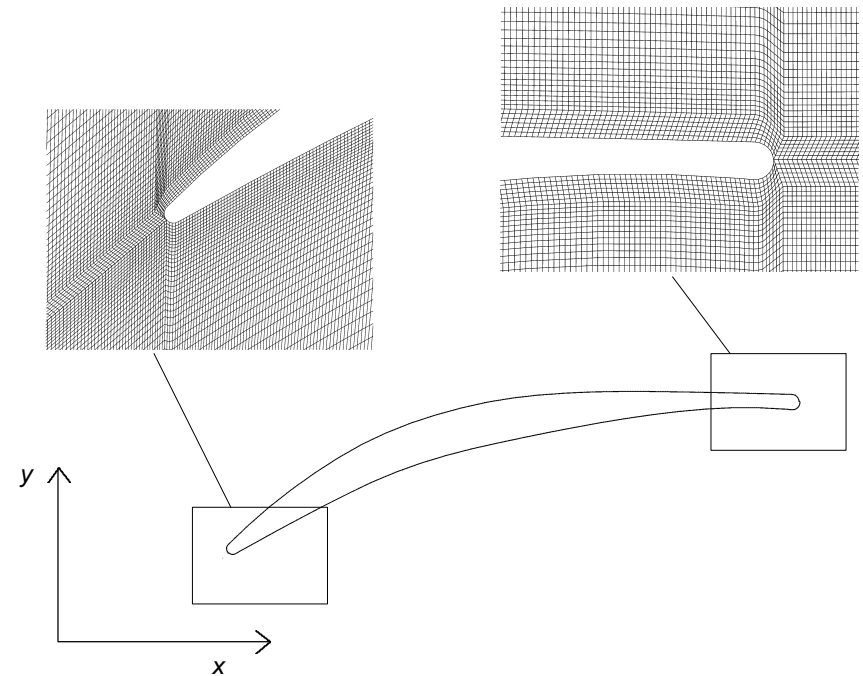
<i>Domains</i>	<i>S(pr)</i>	<i>E(pr)</i>
8	8.00	1.00
16	11.23	0.70
32	16.51	0.50

Non-dimensional streamwise velocity profile along span - mid of outlet bend section experiments (Taylor et al., 1981)

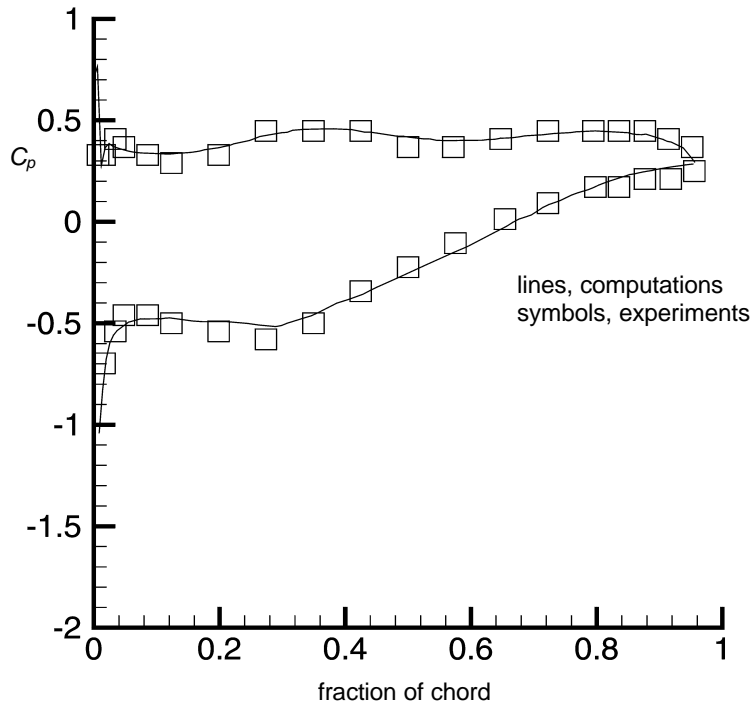
VISCOUS FLOW IN A CONTROLLED DIFFUSION PROFILE CASCADE

- EXPERIMENTS ELAZAR AND SHREEVE (1990)
- $Re=700.000$, $i = 0$ (DESIGN INLET FLOW ANGLE)
- $Ma_{inlet} @ 0.25$
- GRID (46737 POINTS, 577×81)
- IBM SP2

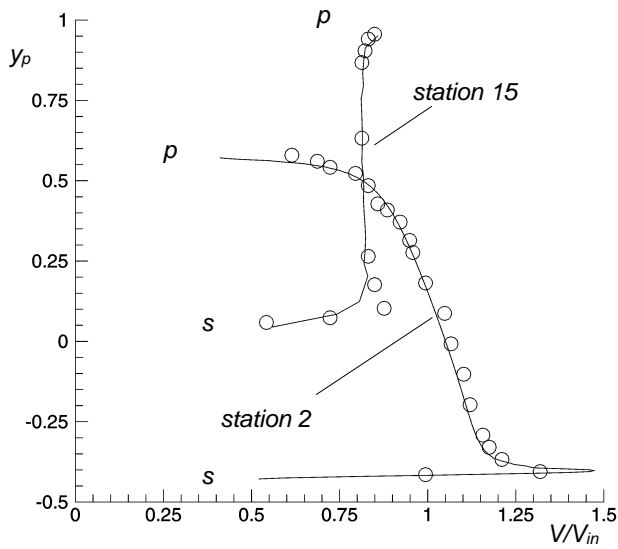
<i>Domains</i>	<i>S(pr)</i>	<i>E(pr)</i>
2	2.00	1.00
4	3.34	0.83
6	4.78	0.80
8	6.60	0.825
12	9.95	0.83



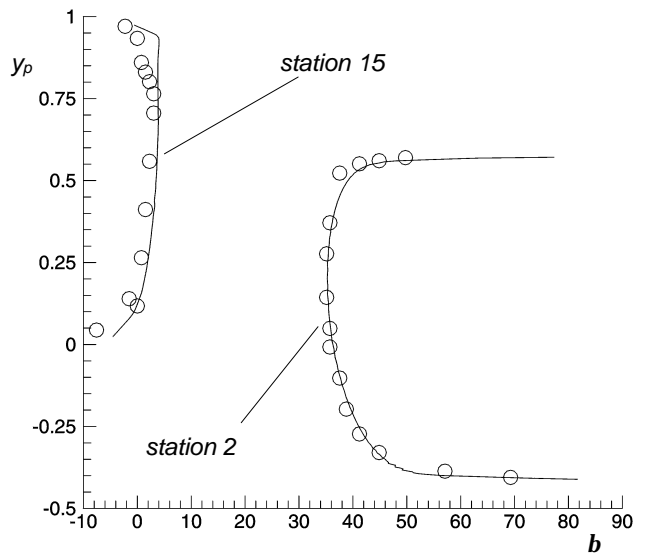
pressure coefficient distributions



streamwise velocity distributions



flow angle distributions

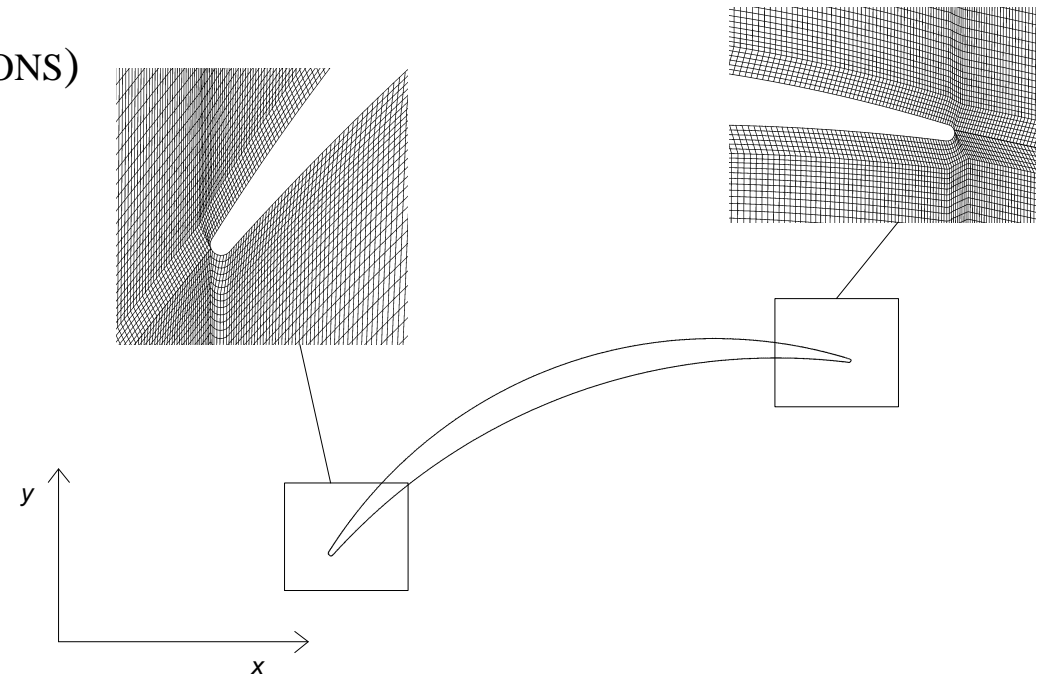


Flow survey at cascade entrance (station 2) and exit (station 15)
(lines: computations, symbols, experiments)

VISCOUS FLOW IN A DCA CASCADE

- EXPERIMENTS ZIERKE AND DEUTCH (1989)
- $Re=501.000$, $i = -1.5^\circ$ (NEAR DESIGN CONDITIONS)
- $Ma_{inlet} @ 0.1$
- GRID (46737 POINTS, 577×81)
- IBM SP2

Domains	$S(pr)$	$E(pr)$
2	2.00	1.00
4	3.06	0.76
6	4.54	0.75
8	6.02	0.75
12	8.28	0.69



DCA GLOBAL PERFORMANCE COEFFICIENTS

	\bar{w}	\bar{C}_{p_2}	\bar{D}
experiments	0.09	0.47	0.55
computation	0.07	0.55	0.58

Pitchwise averaged cascade parameters

TOTAL PRESSURE LOSS COEFF. $\bar{w} = \frac{\overline{p_{01}} - \overline{p_{02}}}{\frac{1}{2} \mathbf{r} \overline{U_1}^2}$

GLOBAL PRESURE COEFF. $\bar{C}_{p_2} = \frac{\overline{p_2} - \overline{p_1}}{\frac{1}{2} \mathbf{r} \overline{U_1}^2}$

DIFFUSION FACTOR $\bar{D} = \left(1 - \frac{\overline{U_2}}{\overline{U_1}} \right) + \left(\frac{\overline{U_{t1}} - \overline{U_{t2}}}{2\mathbf{s}\overline{U_1}} \right)$

